



Digital Control Systems: State Space Control System Design

ELEC 3004: Systems: Signals & Controls

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Lecture 21

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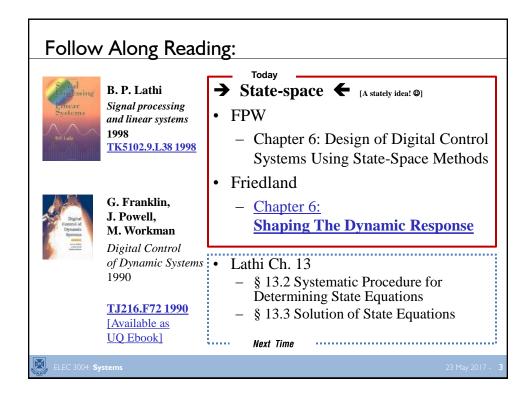
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Lecture Schedule:

Week	Date	Lecture Title
1	28-Feb	Introduction
	2-Mar	Systems Overview
2	7-Mar	Systems as Maps & Signals as Vectors
	9-Mar	Systems: Linear Differential Systems
3	14-Mar	Sampling Theory & Data Acquisition
		Aliasing & Antialiasing
4		Discrete Time Analysis & Z-Transform
	23-Mar	Second Order LTID (& Convolution Review)
5		Frequency Response
	30-Mar	Filter Analysis
6	4-Apr	Digital Filters (IIR) & Filter Analysis
	6-Apr	Digital Filter (FIR)
7		Digital Windows
	13-Apr	FFT
	18-Apr	
	20-Apr	Holiday
	25-Apr	
8	27-Apr	Active Filters & Estimation
9		Introduction to Feedback Control
	4-May	Servoregulation/PID
10	9-May	PID & State-Space
	11-May	State-Space Control
11	16-May	Digital Control Design
	18-May	Stability
12		State Space Control System Design
		Shaping the Dynamic Response
13		System Identification & Information Theory
	1-Jun	Summary and Course Review

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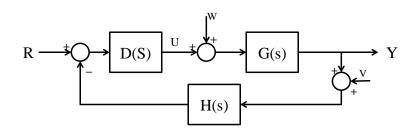


Control Systems Design: tf2ss

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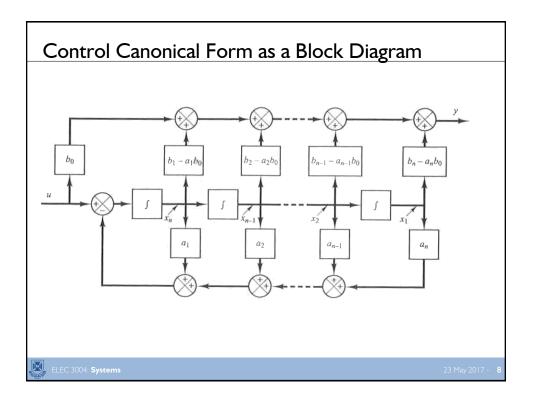
23 May 2017 - 5

Basic Closed-loop Block Diagram



$$TF(s) = \frac{Y(s)}{R(s)} = \frac{D(s)G(s)}{1 + D(s)G(s)H(s)} = \frac{DG}{1 + DGH}$$

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Modal Form

- CCF is not the only way to tf2ss
- Partial-fraction expansion of the system
- → System poles appear as diagonals of Am
- Two issues:
 - The elements of matrix maybe complex if the poles are complex
 - It is non-diagonal with repeated poles



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Modal Form

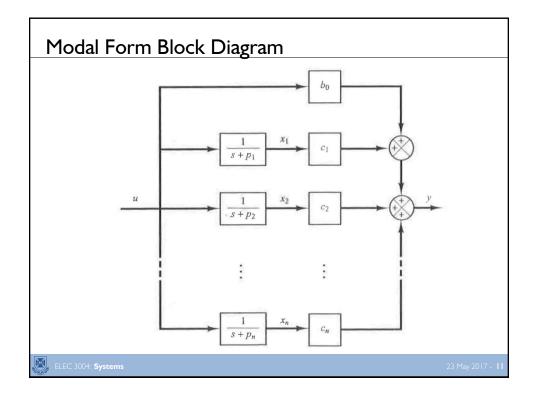
$$\frac{Y(s)}{U(s)} = \frac{b_0 s^n + b_1 s^{n-1} + \dots + b_{n-1} s + b_n}{(s + p_1)(s + p_2) \dots (s + p_n)}$$

$$= b_0 + \frac{c_1}{s + p_1} + \frac{c_2}{s + p_2} + \dots + \frac{c_n}{s + p_n}$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \vdots \\ \vdots \\ \dot{x}_n \end{bmatrix} = \begin{bmatrix} -p_1 \\ -p_2 \\ \vdots \\ 0 \end{bmatrix}$$

$$y = \begin{bmatrix} c_1 \\ c_2 \\ \dots \\ c_n \end{bmatrix} \begin{bmatrix} x_1 \\ 1 \\ \vdots \\ x_n \end{bmatrix} + b_0 u$$
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5



Matlab's tf2ss

- Given: $\frac{Y(s)}{U(s)} = \frac{25.04s + 5.008}{s^3 + 5.03247s^2 + 25.1026s + 5.008}$ Get a state space representation of this system
- Matlab:

• Answer:

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} -5.0325 & -25.1026 & -5.008 \\ 1 & 0 & 0 \\ 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} u$$

$$y = \begin{bmatrix} 0 & 25.04 & 5.008 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \end{bmatrix} u$$

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Control System Design: Obtaining a Time Response

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From SS to Time Response — Impulse Functions

- Given: $\dot{x} = Ax + Bu$
- Solution:

$$\mathbf{x}(t) = e^{\mathbf{A}(t-t_0)}\mathbf{x}(t_0) + \int_{t_0}^t e^{\mathbf{A}(t-\tau)}\mathbf{B}\mathbf{u}(\tau) d\tau$$

- Substituting
$$t_0 = 0$$
 into this:

$$\mathbf{x}(t) = e^{\mathbf{A}t}\mathbf{x}(0-) + \int_{0-}^{t} e^{\mathbf{A}(t-\tau)}\mathbf{B}\mathbf{u}(\tau) d\tau$$

Write the impulse as:

$$\mathbf{u}(t) = \delta(t)\mathbf{w}$$

- where \mathbf{w} is a vector whose components are the magnitudes of \mathbf{r} impulse functions applied at t=0



$$\mathbf{x}(t) = e^{\mathbf{A}t}\mathbf{x}(0-) + \int_{0-}^{t} e^{\mathbf{A}(t-\tau)} \mathbf{B}\delta(\tau) \mathbf{w} \, d\tau$$

$$= e^{\mathbf{A}t}\mathbf{x}(0-) + e^{\mathbf{A}t}\mathbf{B}\mathbf{w}$$



From SS to Time Response — Step Response

- Given: $\dot{x} = Ax + Bu$
- Start with u(t) = k

Where \mathbf{k} is a vector whose components are the magnitudes of r step functions applied at t=0.

$$\mathbf{x}(t) = e^{\mathbf{A}t}\mathbf{x}(0) + \int_0^t e^{\mathbf{A}(t-\tau)}\mathbf{B}\mathbf{k} d\tau$$

$$= e^{\mathbf{A}t}\mathbf{x}(0) + e^{\mathbf{A}t} \left[\int_0^t \left(\mathbf{I} - \mathbf{A}\tau + \frac{\mathbf{A}^2\tau^2}{2!} - \cdots \right) d\tau \right] \mathbf{B}\mathbf{k}$$

$$= e^{\mathbf{A}t}\mathbf{x}(0) + e^{\mathbf{A}t} \left[\mathbf{I}t - \frac{\mathbf{A}t^2}{2!} + \frac{\mathbf{A}^2t^3}{3!} - \cdots \right] \mathbf{B}\mathbf{k}$$

- Assume A is non-singular



$$\mathbf{x}(t) = e^{\mathbf{A}t}\mathbf{x}(0) + e^{\mathbf{A}t}[-(\mathbf{A}^{-1})(e^{-\mathbf{A}t} - \mathbf{I})]\mathbf{B}\mathbf{k}$$
$$= e^{\mathbf{A}t}\mathbf{x}(0) + \mathbf{A}^{-1}(e^{\mathbf{A}t} - \mathbf{I})\mathbf{B}\mathbf{k}$$



23 May 2017 - 15

From SS to Time Response — Ramp Response

- Given: $\dot{x} = Ax + Bu$
- Start with u(t) = tv

Where \mathbf{v} is a vector whose components are magnitudes of ramp functions applied at t = 0

$$\mathbf{x}(t) = e^{\mathbf{A}t}\mathbf{x}(0) + \int_0^t e^{\mathbf{A}(t-\tau)}\mathbf{B}\tau \mathbf{v} d\tau$$

$$= e^{\mathbf{A}t}\mathbf{x}(0) + e^{\mathbf{A}t} \int_0^t e^{-\mathbf{A}\tau}\tau d\tau \mathbf{B}\mathbf{v}$$

$$= e^{\mathbf{A}t}\mathbf{x}(0) + e^{\mathbf{A}t} \left(\frac{1}{2}t^2 - \frac{2\mathbf{A}}{3!}t^3 + \frac{3\mathbf{A}^2}{4!}t^4 - \frac{4\mathbf{A}^3}{5!}t^5 + \cdots\right) \mathbf{B}\mathbf{v}$$

- Assume A is non-singular



$$\mathbf{x}(t) = e^{\mathbf{A}t}\mathbf{x}(0) + (\mathbf{A}^{-2})(e^{\mathbf{A}t} - \mathbf{I} - \mathbf{A}t)\mathbf{B}\mathbf{v}$$
$$= e^{\mathbf{A}t}\mathbf{x}(0) + [\mathbf{A}^{-2}(e^{\mathbf{A}t} - \mathbf{I}) - \mathbf{A}^{-1}t]\mathbf{B}\mathbf{v}$$



• Given:
$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_1 \end{bmatrix} = \begin{bmatrix} -1 & -0.5 \\ 1 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0.5 \\ 0 \end{bmatrix} u, \quad \begin{bmatrix} x_1(0) \\ x_2(0) \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$
• Solution:
$$\mathbf{A} = \begin{bmatrix} -1 & -0.5 \\ 1 & 0 \end{bmatrix}, \quad \mathbf{B} = \begin{bmatrix} 0.5 \\ 0 \end{bmatrix} \qquad \mathbf{\Phi}(t) = e^{\mathbf{A}t} = \mathcal{L}^{-1}[(s\mathbf{I} - \mathbf{A})^{-1}]$$

$$(s\mathbf{I} - \mathbf{A})^{-1} = \begin{bmatrix} s+1 & 0.5 \\ -1 & s \end{bmatrix}^{-1} = \frac{1}{s^2 + s + 0.5} \begin{bmatrix} s & -0.5 \\ 1 & s + 1 \end{bmatrix} \qquad \mathbf{\Phi}(t) = e^{\mathbf{A}t} = \mathcal{L}^{-1}[(s\mathbf{I} - \mathbf{A})^{-1}]$$

$$= \begin{bmatrix} s+0.5 - 0.5 \\ (s+0.5)^2 + 0.5^2 \end{bmatrix} \xrightarrow{s+0.5 + 0.5} \begin{bmatrix} s & -0.5 \\ 1 & s + 1 \end{bmatrix} \qquad \mathbf{\Phi}(t) = e^{\mathbf{A}t} = \mathcal{L}^{-1}[(s\mathbf{I} - \mathbf{A})^{-1}]$$

$$= \begin{bmatrix} \frac{s+0.5 - 0.5}{(s+0.5)^2 + 0.5^2} & \frac{-0.5}{(s+0.5)^2 + 0.5^2} \\ \frac{1}{(s+0.5)^2 + 0.5^2} & \frac{s+0.5 + 0.5}{(s+0.5)^2 + 0.5^2} \end{bmatrix}$$

$$- \mathbf{Set} \ \mathbf{k} = \mathbf{1}, \ \mathbf{x}(\mathbf{0}) = \mathbf{0};$$

$$\mathbf{x}(t) = e^{\mathbf{A}t}(0) + \mathbf{A}^{-1}(e^{\mathbf{A}t} - 1)\mathbf{B}k$$

$$= \mathbf{A}^{-1}(e^{\mathbf{A}t} - 1)\mathbf{B}$$

$$= \begin{bmatrix} 0 & 1 \\ -2 & -2 \end{bmatrix} \begin{bmatrix} 0.5e^{-0.5t}(\cos 0.5t - \sin 0.5t) - 0.5 \\ e^{-0.5t}(\sin 0.5t) \end{bmatrix} \qquad \mathbf{y}(t) = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = x_1 = e^{-0.5t}\sin 0.5t$$

Example II: Obtain the Step Response

• Given:

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$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u$$

 $= \begin{bmatrix} e^{-0.5t} \sin 0.5t \\ -e^{-0.5t} (\cos 0.5t + \sin 0.5t) + 1 \end{bmatrix}$

$$u(t) = 1(t)$$

• Solution:

$$\mathbf{A} = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}, \quad \mathbf{B} = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$



$$\mathbf{\Phi}(t) = e^{\mathbf{A}t} = \mathcal{L}^{-1}[(s\mathbf{I} - \mathbf{A})^{-1}]$$

$$\Phi(t) = e^{\mathbf{A}t} = \begin{bmatrix} 2e^{-t} - e^{-2t} & e^{-t} - e^{-2t} \\ -2e^{-t} + 2e^{-2t} & -e^{-t} + 2e^{-2t} \end{bmatrix}$$

$$\mathbf{x}(t) = e^{At}\mathbf{x}(0) + \int_{0}^{t} \begin{bmatrix} 2e^{-(t-\tau)} - e^{-2(t-\tau)} & e^{-(t-\tau)} - e^{-2(t-\tau)} \\ -2e^{-(t-\tau)} + 2e^{-2(t-\tau)} - e^{-(t-\tau)} + 2e^{-2(t-\tau)} \end{bmatrix} \begin{bmatrix} 0 \\ 1 \end{bmatrix} [1] d\tau$$

$$\begin{bmatrix} x_{1}(t) \\ x_{2}(t) \end{bmatrix} = \begin{bmatrix} 2e^{-t} - e^{-2t} & e^{-t} - e^{-2t} \\ -2e^{-t} + 2e^{-2t} - e^{-t} + 2e^{-2t} \end{bmatrix} \begin{bmatrix} x_{1}(0) \\ x_{2}(0) \end{bmatrix} + \begin{bmatrix} \frac{1}{2} - e^{-t} + \frac{1}{2}e^{-2t} \\ e^{-t} - e^{-2t} \end{bmatrix}$$

- Assume x(0)=0:

$$\begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix} = \begin{bmatrix} 2e^{-t} - e^{-2t} & e^{-t} - e^{-2t} \\ -2e^{-t} + 2e^{-2t} & -e^{-t} + 2e^{-2t} \end{bmatrix} \begin{bmatrix} x_1(0) \\ x_2(0) \end{bmatrix} + \begin{bmatrix} \frac{1}{2} - e^{-t} + \frac{1}{2}e^{-2t} \\ e^{-t} - e^{-2t} \end{bmatrix}$$

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Break © ELEC 3004: Systems 23 May 2017 - 19

The Direct Method of Digital Controls –

NOT to be confused with **Controller Emulation** (e.g., Tustin's Method)

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Direct Design Method Of Ragazzini (See also: FPW 5.7 pp.216-222)

Start with 3 Discrete Transfer Functions:

- G(z): TF¹ of a **plant** + a **hold** (e.g., from a ZOH)
- **D**(**z**): A **controller** TF to do the job (what we want here)
- **H**(**z**): The final desired TF between **R** (reference) and **Y** (output)
- Thus²:

$$H(z) = \frac{DG}{1 + DG}$$
$$\Rightarrow D(z) = \frac{1}{G} \frac{H}{1 - H}$$

- This calls for a D(z) that will cancel the plant effects and that will add whatever is
 necessary to give the desired result. The problem is to discover and implement
 constraints on H(z) so that we do not ask for the impossible.
 - This implies that we **need some constraints** on both $\mathbf{H}(\mathbf{z})$ and $\mathbf{D}(\mathbf{z})$
- 1: Transfer Function
- 2: Mental Quiz: What does **1+DG** say about the sign of the feedback (positive or negative)? That is, what is the characteristic equation for a system with positive feedback?



23 May 2017 - **2**

Direct Design Method Of Ragazzini [2]: Design Constraints: I. Causality

- Remember/Recall an Interesting Point:
 - From z-transform theory we know that if D(z) is causal,
 then as z → ∞ its transfer function is well behaved
 it does not have a pole at infinity.
- $D(z) = \frac{1}{G} \frac{H}{1-H}$ implies that if G(z) = 0 (at ∞), then D(z) would have a pole (at ∞) unless H(z) cancels it.

 $\mathbf{H}(\mathbf{z})$ must have a zero (at ∞) of the same order as $\mathbf{G}(\mathbf{z})$'s 0s (at ∞)

 \rightarrow Which means: If there is a lag in the plant (G(z) starts with z^{-l}) then causality requires that the delay of H(z) is that the closed-loop system must be at least as long a delay of the plant.

(Whoa! It might sound deep, but it's rather intuitive ©)



22 Mar. 2017 22

Direct Design Method Of Ragazzini [3]: Design Constraints: II. Stability

• The characteristic equation and the closed loop roots:

$$1 + D(z)G(z) = 0$$

- Define³ $D = \frac{c}{d}$ and $G = \frac{b}{a} \implies ad + bc = 0$
- Define $z \alpha$ as a pole of G(z) and a <u>common factor</u> in DG that represents D(z) cancelling a pole/zero of G(z).
- Then this common factor remains a factor of the characteristic polynomial.
- If this factor is outside the unit circle, then the system is unstable!

1-H(z) must contain as zeros all the poles of G(z) that are outside the unit circle & H(z) must contain as zeros all the zeros of G(z) that are outside the unit circle

3: Note the switching of the "alphabetical-ness" of these two fractions



23 May 2017 - **2**3

Direct Design Method Of Ragazzini [4]: Design Constraints: III. Steady State Accuracy

• The error from $\mathbf{H}(\mathbf{z})$ is given by:

$$E(z) = R(z)(1 - H(z))$$

• If the system is "Type 1" (with a constant velocity/first derivative (K_v) – Then⁴ $E_{SS}^{Step} = 0$ and $E_{SS}^{Ramp} = \frac{1}{K_v}$

$$H(z) = 1$$
&
$$-T_s \frac{dH(z)}{d(z)} \Big|_{z=1} = \frac{1}{K_v} H(z) = 1$$

4: E_{ss}: steady-state error



22 Mar. 2017 - 24

Direct Design Method Of Ragazzini [5]: An Example

- Consider the plant: $s^2 + s + 1 = 0$ With $T_s=1 \rightarrow z$ -Transform: $z^2 + 0.786z + 0.368=0$
- Let's design this system such that
 - $-K_{v}=1$
 - Poles at the roots of the plant equation & additional poles as needed

$$\Rightarrow H(z) = \frac{b_0 + b_1 z^{-1} + b_2 z^{-2} + b_3 z^{-3} + \dots}{1 - 0.786 z^{-1} + 0.368 z^{-2}}$$

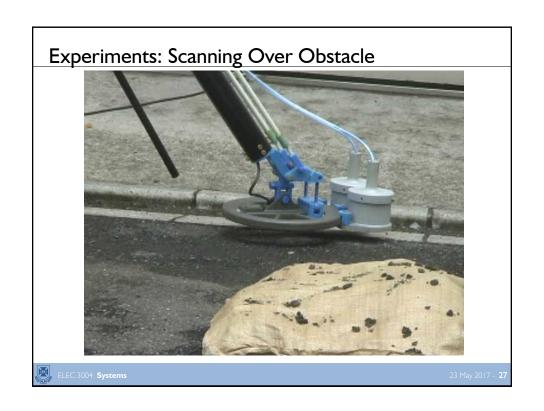
- I. Causality: $H(z)|_{z=\infty} = 0 \rightarrow b_0 = 0$
- II. Stability: All poles/zeros of G(z) are in the unit circle
 - except for b_0 , which is taken care of by $b_0 = [Const] = 0$
- III. Tracking:
 - $H(1) = b_1 + b_2 + b_3 + \dots = 1 \cdot (1 0.786 + 0.368) &$
 - $-\{1\} \frac{dH(z)}{d(z^{-1})} \bigg|_{z=1} = \frac{1}{\{1\}} \implies \frac{b_1 + 2b_2 + 3b_3 + \dots [-.05014]}{(1 0.786 + 0.368)}$ (note the z^{-1})
 - Truncate the number of unknowns to 2 "zeros" ... thus solve for b_1 and b_2 (& set $b_3,b_4,...=0$)

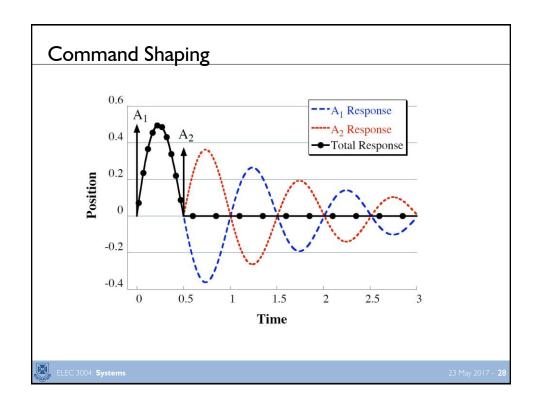
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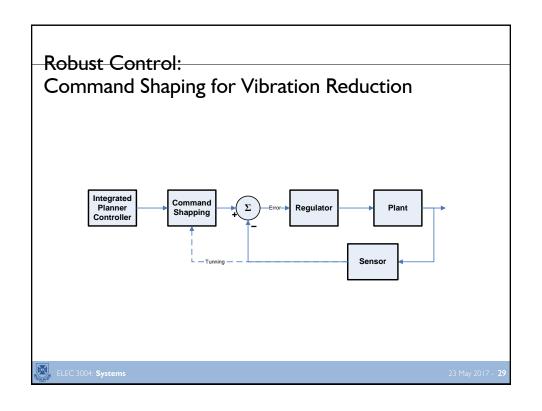
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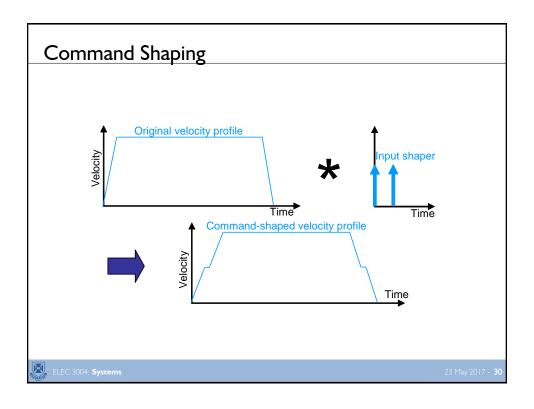
Application Example 1: Command Shaping

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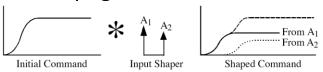








Command Shaping



• Zero Vibration (ZV)

$$\begin{bmatrix} A_i \\ t_i \end{bmatrix} = \begin{bmatrix} \frac{1}{1+K} & \frac{K}{1+K} \\ 0 & \frac{T_d}{2} \end{bmatrix}$$

$$K = e^{\left(\frac{-\zeta\pi}{\sqrt{1-\zeta^2}}\right)}$$

• Zero Vibration and Derivative (ZVD)

$$\begin{bmatrix} A_i \\ t_i \end{bmatrix} = \begin{bmatrix} \frac{1}{(1+K)^2} & \frac{2K}{(1+K)^2} & \frac{K^2}{(1+K)^2} \\ 0 & \frac{T_d}{2} & T_d \end{bmatrix}$$

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23 May 2017 - **3**





16

Next Time...



- Digital Feedback Control
- Review:
 - Chapter 2 of FPW
- More Pondering??

