



Discrete (Digital) Feedback Control: PID, Regulators, Compensators, Computer Control, etc.

ELEC 3004: **Systems**: Signals & Controls
Dr. Surya Singh

Lecture 20

elec3004@itee.uq.edu.au

<http://robotics.itee.uq.edu.au/~elec3004/>

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	8-Mar	System Models
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Goals for the Week

- Stability of Digital Systems
- Lead/Lag Compensators
- ZOH and Discretisation



Two classes of control design

The system...

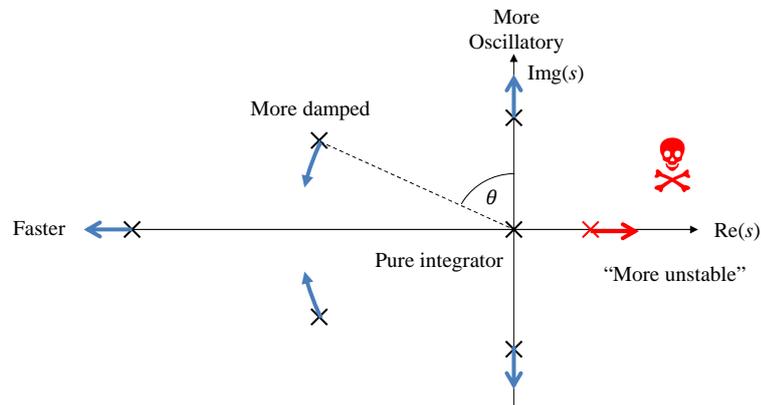
- Isn't fast enough
- Isn't damped enough
- Overshoots too much
- Requires too much control action
(“Performance”)

- Attempts to spontaneously disassemble itself
(“Stability”)



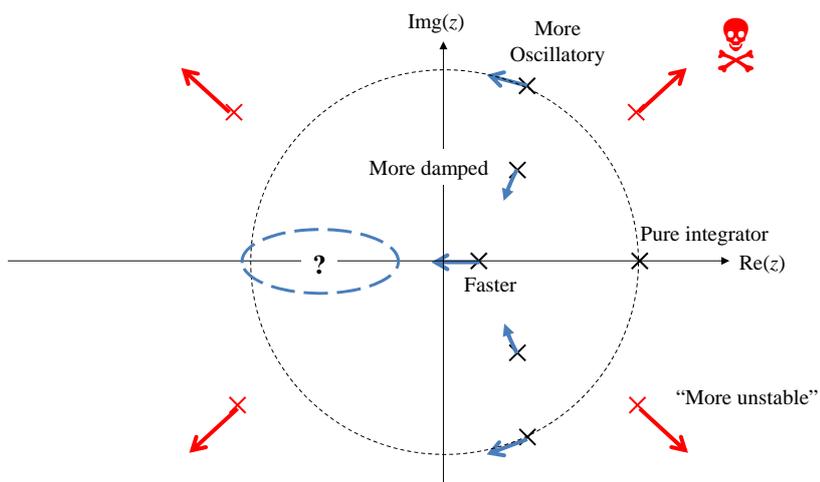
Recall dynamic responses

- Moving pole positions change system response characteristics



Recall dynamic responses

- Ditto the z-plane:



The fundamental control problem

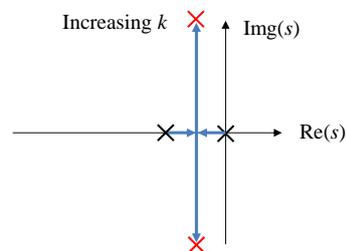
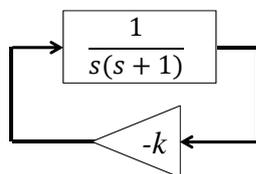
The poles are in the wrong place

How do we get them where we want them to be?



Recall the root locus

- We know that under feedback gain, the poles of the closed-loop system move
 - The root locus tells us where they go!
 - We can solve for this analytically*

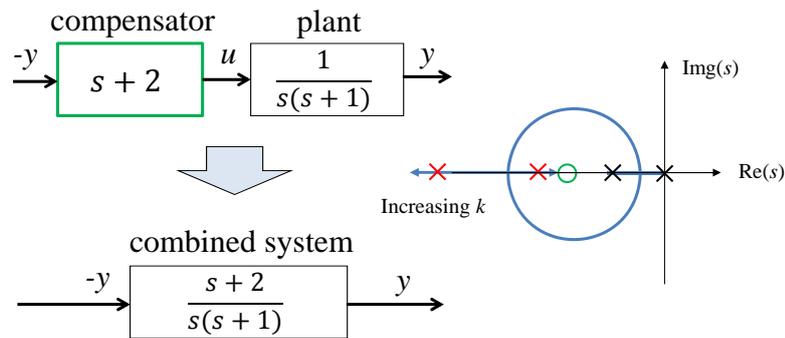


Root loci can be plotted for all sorts of parameters, not just gain!



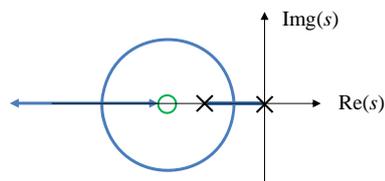
Dynamic compensation

- We can do more than just apply gain!
 - We can add dynamics into the controller that alter the open-loop response



But what dynamics to add?

- Recognise the following:
 - A root locus starts at poles, terminates at zeros
 - “Holes eat poles”
 - Closely matched pole and zero dynamics cancel
 - The locus is on the real axis to the left of an odd number of poles (treat zeros as ‘negative’ poles)



Some standard approaches

- Control engineers have developed time-tested strategies for building compensators
- Three classical control structures:
 - Lead
 - Lag
 - Proportional-Integral-Derivative (PID)
(and its variations: P, I, PI, PD)

How do they work?



Lead/lag compensation

- Serve different purposes, but have a similar dynamic structure:

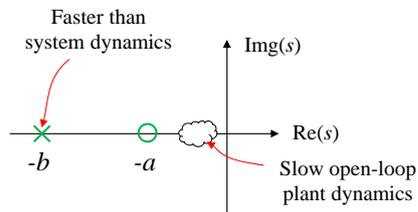
$$D(s) = \frac{s + a}{s + b}$$

Note:

Lead-lag compensators come from the days when control engineers cared about constructing controllers from networks of op amps using frequency-phase methods. These days pretty much everybody uses PID, but you should at least know what the heck they are in case someone asks.



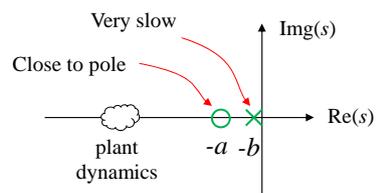
Lead compensation: $a < b$



- Acts to decrease rise-time and overshoot
 - Zero draws poles to the left; adds phase-lead
 - Pole decreases noise
- Set a near desired ω_n ; set b at ~ 3 to $20 \times a$



Lag compensation: $a > b$



- Improves steady-state tracking
 - Near pole-zero cancellation; adds phase-lag
 - Doesn't break dynamic response (too much)
- Set b near origin; set a at ~ 3 to $10 \times b$



PID – the Good Stuff

- Proportional-Integral-Derivative control is the control engineer's hammer*
 - For P,PI,PD, etc. just remove one or more terms

$$C(s) = k \left(1 + \frac{1}{\tau_i s} + \tau_d s \right)$$

Proportional
Integral
Derivative

*Everything is a nail. That's why it's called "Bang-Bang" Control ☺

PID – the Good Stuff

- PID control performance is driven by three parameters:
 - k : system gain
 - τ_i : integral time-constant
 - τ_d : derivative time-constant

You're already familiar with the effect of gain.
What about the other two?

Integral

- Integral applies control action based on accumulated output error
 - Almost always found with P control
- Increase dynamic order of signal tracking
 - Step disturbance steady-state error goes to zero
 - Ramp disturbance steady-state error goes to a constant offset

Let's try it!



Integral

- Consider a first order system with a constant load disturbance, w ; (recall as $t \rightarrow \infty, s \rightarrow 0$)

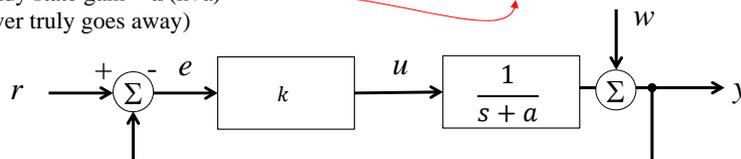
$$y = k \frac{1}{s+a} (r - y) + w$$

$$(s+a)y = k(r-y) + (s+a)w$$

$$(s+k+a)y = kr + (s+a)w$$

$$y = \frac{k}{s+k+a} r + \frac{(s+a)}{s+k+a} w$$

Steady state gain = $a/(k+a)$
(never truly goes away)



Now with added integral action

$$y = k \left(1 + \frac{1}{\tau_i s} \right) \frac{1}{s + a} (r - y) + w$$

$$y = k \frac{s + \tau_i^{-1}}{s} \frac{1}{s + a} (r - y) + w$$

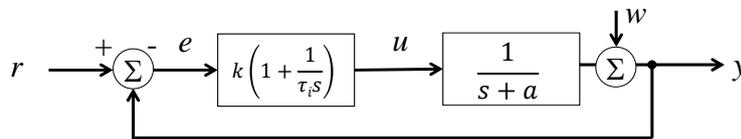
Same dynamics

$$s(s + a)y = k(s + \tau_i^{-1})(r - y) + s(s + a)w$$

$$(s^2 + (k + a)s + \tau_i^{-1})y = k(s + \tau_i^{-1})r + s(s + a)w$$

$$y = \frac{k(s + \tau_i^{-1})}{(s^2 + (k + a)s + \tau_i^{-1})} r + \frac{s(s + a)}{k(s + \tau_i^{-1})} w$$

Must go to zero for constant w!



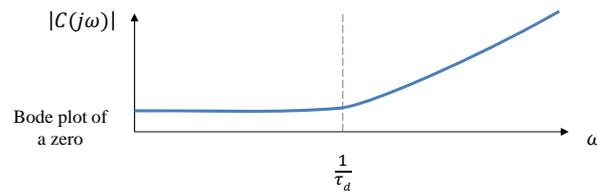
Derivative

- Derivative uses the rate of change of the error signal to anticipate control action
 - Increases system damping (when done right)
 - Can be thought of as ‘leading’ the output error, applying correction predictively
 - Almost always found with P control*

*What kind of system do you have if you use D, but don't care about position? Is it the same as P control in velocity space?

Derivative

- It is easy to see that PD control simply adds a zero at $s = -\frac{1}{\tau_d}$ with expected results
 - Decreases dynamic order of the system by 1
 - Absorbs a pole as $k \rightarrow \infty$
- Not all roses, though: derivative operators are sensitive to high-frequency noise



PID

- Collectively, PID provides two zeros plus a pole at the origin
 - Zeros provide phase lead
 - Pole provides steady-state tracking
 - Easy to implement in microprocessors
- Many tools exist for optimally tuning PID
 - Zeigler-Nichols
 - Cohen-Coon
 - Automatic software processes

Be alert

- If gains and time-constants are chosen poorly, all of these compensators can induce oscillation or instability.
- However, when used properly, PID can stabilise even very complex unstable third-order systems



Now in discrete

- Naturally, there are discrete analogs for each of these controller types:

Lead/lag: $\frac{1-\alpha z^{-1}}{1-\beta z^{-1}}$

PID: $k \left(1 + \frac{1}{\tau_i(1-z^{-1})} + \tau_d(1-z^{-1}) \right)$

But, where do we get the control design parameters from?
The s-domain?



Emulation vs Discrete Design

- Remember: polynomial algebra is the same, whatever symbol you are manipulating:

$$\text{eg. } s^2 + 2s + 1 = (s + 1)^2$$

$$z^2 + 2z + 1 = (z + 1)^2$$

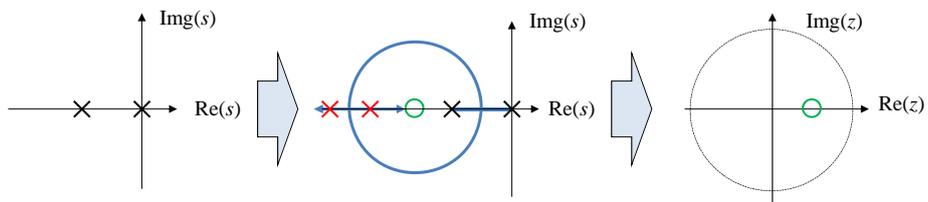
Root loci behave the same on both planes!

- Therefore, we have two choices:
 - Design in the s-domain and digitise (emulation)
 - Design only in the z-domain (discrete design)



Emulation design process

1. Derive the dynamic system model ODE
2. Convert it to a continuous transfer function
3. Design a continuous controller
4. Convert the controller to the z-domain
5. Implement difference equations in software



Emulation design process

- Handy rules of thumb:
 - Use a sampling period of 20 to 30 times faster than the closed-loop system bandwidth
 - Remember that the sampling ZOH induces an effective $T/2$ delay
 - There are several approximation techniques:
 - Euler's method
 - Tustin's method
 - Matched pole-zero
 - Modified matched pole-zero



Tustin's method

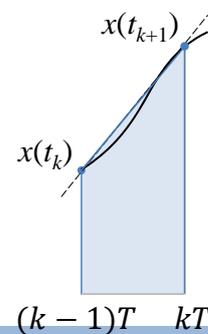
- Tustin uses a trapezoidal integration approximation (compare Euler's rectangles)
- Integral between two samples treated as a straight line:

$$u(kT) = \frac{T}{2} [x(k-1) + x(k)]$$

Taking the derivative, then z-transform yields:

$$S = \frac{2z-1}{Tz+1}$$

which can be substituted into continuous models



Matched pole-zero

- If $z = e^{sT}$, why can't we just make a direct substitution and go home?

$$\frac{Y(s)}{X(s)} = \frac{s+a}{s+b} \Rightarrow \frac{Y(z)}{X(z)} = \frac{z-e^{-aT}}{z-e^{-bT}}$$

- Kind of!
 - Still an approximation
 - Produces quasi-causal system (hard to compute)
 - Fortunately, also very easy to calculate.



Matched pole-zero

The process:

1. Replace continuous poles and zeros with discrete equivalents:

$$(s + a) \Rightarrow (z - e^{-aT})$$

2. Scale the discrete system DC gain to match the continuous system DC gain
3. If the order of the denominator is higher than the numerator, multiply the numerator by $(1 - z^{-1})$ until they are of equal order*

* This introduces an averaging effect like Tustin's method



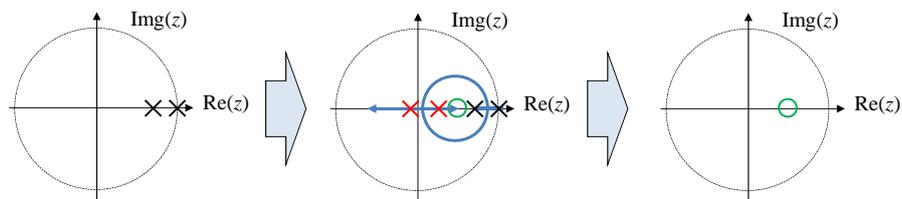
Modified matched pole-zero

- We'd prefer it if we didn't require instant calculations to produce timely outputs
- Modify step 2 to leave the dynamic order of the numerator one less than the denominator
 - Can work with slower sample times, and at higher frequencies



Discrete design process

1. Derive the dynamic system model ODE
2. Convert it to a discrete transfer function
3. Design a digital compensator
4. Implement difference equations in software
5. Platypus Is Divine!



Discrete design process

- Handy rules of thumb:
 - Sample rates can be as low as twice the system bandwidth
 - but 5 to 10× for “stability”
 - 20 to 30 × for better performance
 - A zero at $z = -1$ makes the discrete root locus pole behaviour more closely match the s-plane
 - Beware “dirty derivatives”
 - dy/dt terms derived from sequential digital values are called ‘dirty derivatives’ – these are especially sensitive to noise!
 - Employ actual velocity measurements when possible



Announcements:



- Final Exam:
 - 15 Questions (60% Short Answer, 40% Regular Problems)
 - 3 Hours
 - Closed-book
 - Took tutor ~90min to complete
 - Equation sheet will be provided (in addition to your own)
[See Prac Final – Coming out next week]
 - Yes, it has an unexpected twist at the end, but you’ll like it. 😊

→ Saturday, June 15 at 9:30 AM (sorry!)

ELEC3004	Signals, Systems & Control	15/06/2013	9:30 AM	Agarwal, Atu - Vasuian, Fab	UQ Union Complex (21) - Innes Room
				Vermeulen, Nat - Zulkarnain, Moh	UQ Union Complex (21) - Heath Room

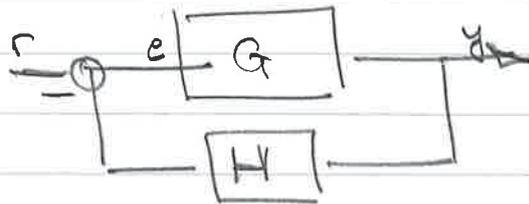


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→ feedback



$$y = G(e)$$

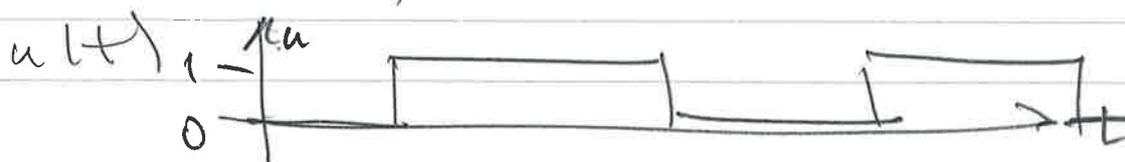
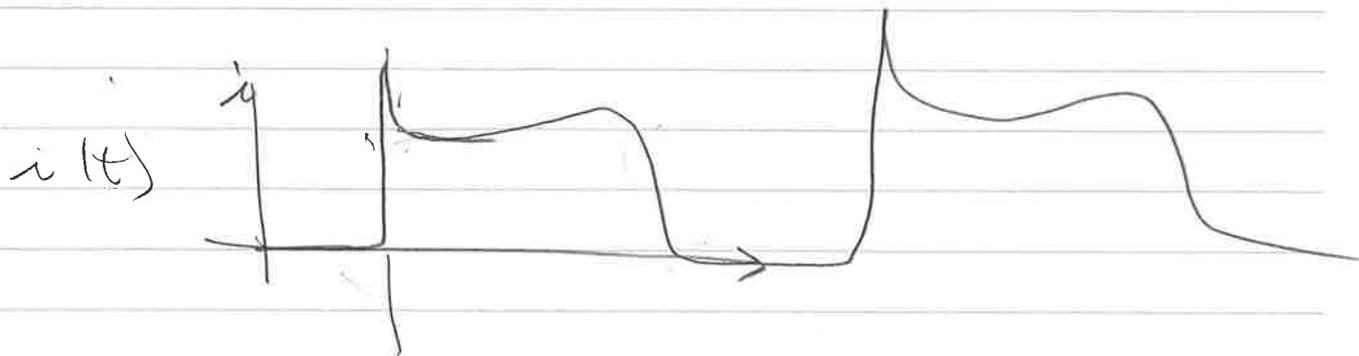
$$\frac{y}{r} = \frac{G}{1+GH}$$

what's bang-bang control

$$u(t) = \begin{cases} 1 & \text{for certain conditions} \\ 0 & \text{else} \end{cases}$$

Fridge:

$$u_f(t) = \begin{cases} 1 & \text{if } T > T^* \\ 0 & \text{if } T \leq T^* \end{cases}$$

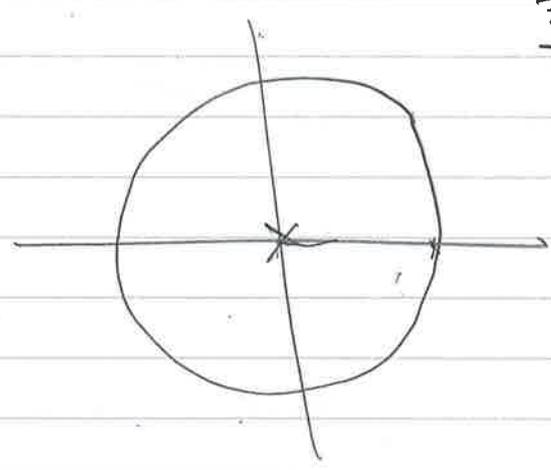


→ D controller + noise

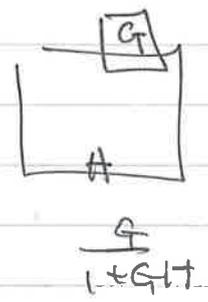
$$y(t) = \sin(\omega t) + 0.01 \cdot \sin(100\omega t)$$

$$y'(t) = \omega [\cos(\omega t) + 100 \cdot (0.01) \cdot \cos(100\omega t)]$$

$$y''(t) = -\omega^2 [\sin(\omega t) + 100^2 (0.01) \cdot \sin(\omega t)]$$



ω



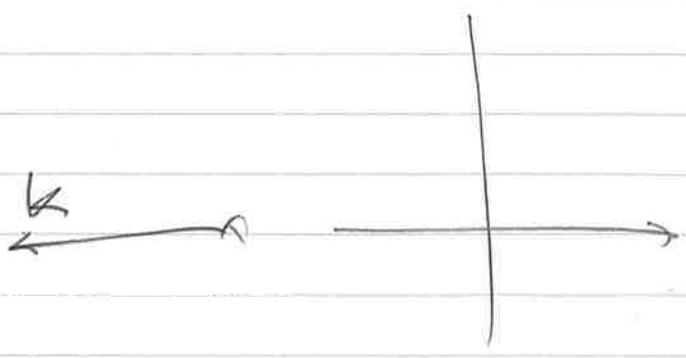
Hz

$$\frac{G}{1+GH}$$

$$1+kH=0$$

s

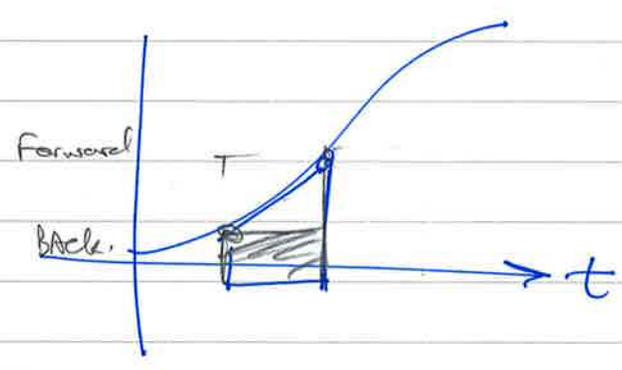
$$\leftarrow \frac{0.1052}{z-1.105}$$



→ Numerical integration

$$\frac{y(s)}{x(s)} = H(s) = \frac{a}{s+a}$$

$$\dot{y} + ay = ax$$



Trapezoidal

$$H_T = \frac{a}{\left(\frac{2}{T}\right) \left[\frac{z-1}{z+1} \right] + a}$$

Forward :

$$\frac{z-1}{T}$$

Backward: $\frac{z-1}{Tz}$

TRAP $\left[\frac{2}{T} \cdot \left(\frac{z-1}{z+1} \right) \right]$